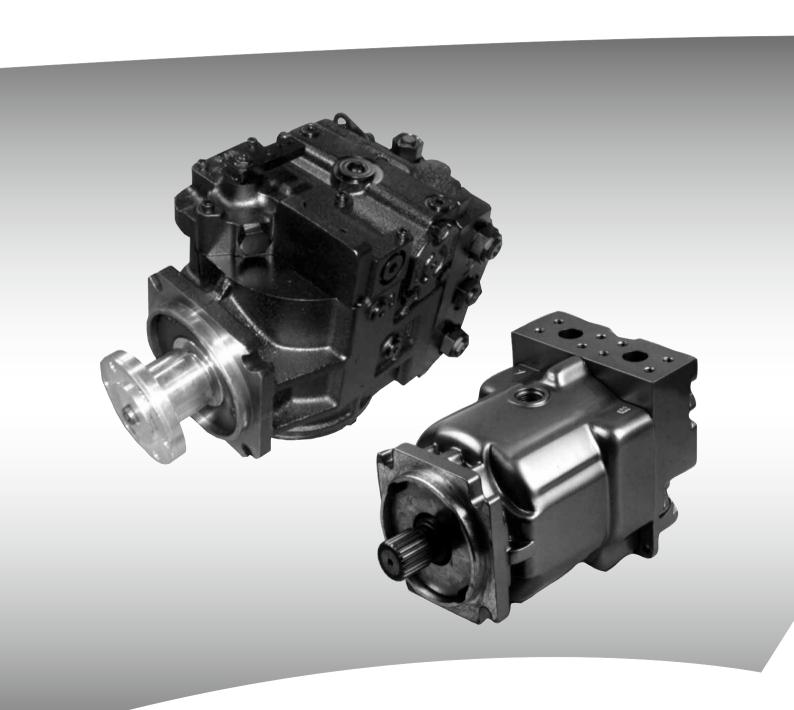


Technical Information

Series T90

Transit Mixer Drive System





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General Description

General

The new Transit Mixer Drive System from Danfoss is based on more than 30 years' experience in the worldwide use of drum drive systems in transit mixers. Innovative electronics suitable for mobile use, combined with reliable technology, are the result of this development.

Danfoss offers, as a single supplier, complete systems for drum sizes from **6-18m³** [8-24 yd³] from one source.

The smart system resets the standard relating to the market requirements for reliability and simple handling.

Product Features

- Hydrostatic transmission
 - overall size 055, 075 and 100 cm³ [3.35, 4.49 and 6.10 in³]
 - rotational group proven over millions in units
 - noise reduction by 12 dB (A)
 - exemplary reductions in overall volume and weight
- Operating units
 - external and cab-mounted station
 - simple installation and wiring
 - electric plug connections suitable for mobile use
 - no adjustments, "Plug and Perform"®
- direction of rotation and speed via latched rotary switch on the cab-mounted or external station
- identical operating elements on the cab-mounted and external stations
- external station pushbutton: STOP/START the drum
- external station pushbutton: STOP/START the drum with the cab-mounted station active
- **System Features** cab-mounted station pushbutton: change over to external station and vice versa
 - status reporting from the cab-mounted and external stations via LEDs
 - active cab-mounted station during transit means the external station is switched off
 - constant drum speed with variable pump-drive speed
 - constant drum speed, irrespective of the loading of the drum
 - exact repeatability of the drum speed by means of a latched rotary switch
 - automatic maximum drum-speed limitation

A convincing, reliable and smart system solution from Danfoss which (at present) is still missing from your transit mixer?

Contact us! Our worldwide sales organization is ready to serve you.



General Description

Series T90 Family of Pumps

Danfoss provides **Series T90** as an advanced type of axial piston variable displacement pumps for concret mixers, the development of which is based on more than 30 years of our experience in applying our products in the global market. The new T90 axial piston variable displacement pumps are derived from the sophisticated earlier type of S90 pumps, and are suitable for extended concrete mixer applications.

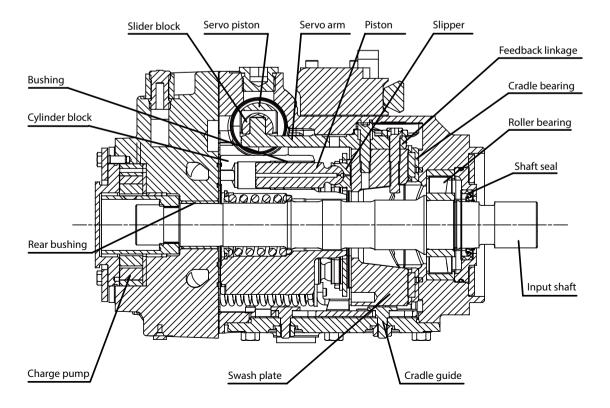
Series T90 variable displacement pumps are compact, high power density units. All models utilize the parallel axial piston/slipper concept in conjunction with a tiltable swashplate to vary the pump's displacement. Reversing the angle of the swashplate reverses the flow of oil from the pump and thus reverses the direction of rotation of the motor output.

Series T90 pumps include an integral charge pump which is manually controlled to provide system replenishing and cooling oil flow, as well as control fluid flow.

- Series T90 axial piston pumps are designed with the most advanced technology
- With optional sizes 055, 075, 100
- Installation: SAE standard flange
- Axial piston design of high effeciency
- Proved reliability and excellent performance
- Compact, light weight
- Worldwide sales and services
- Metric standard threads for main ports (A and B)

Design

Series T90 pump cross-section

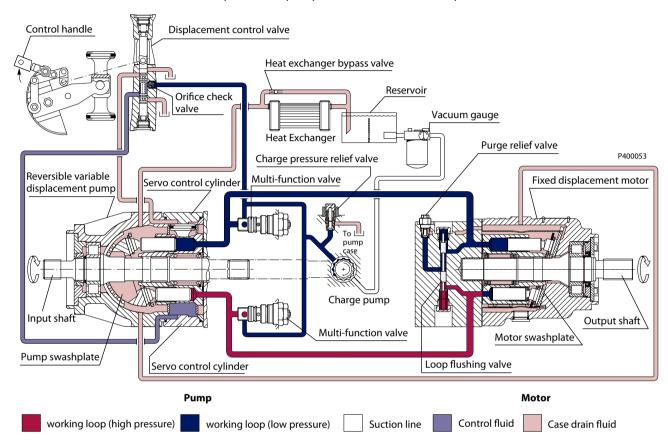




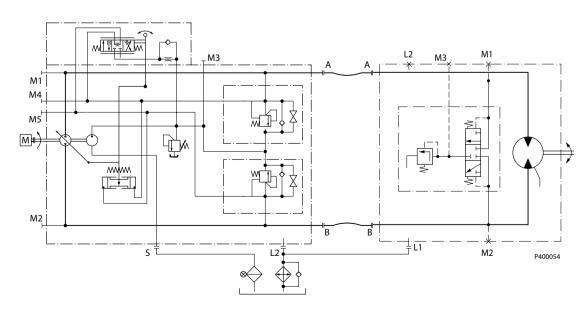
General Description

Pictorial Circuit Diagram

This configuration shows a hydrostatic transmission using a Series T90 axial piston variable displacement pump and a Series 90 fixed displacement motor.



System Schematic





Technical Specifications

Features

Feature	Unit	055	075	100	
Disalagament	cm ³	55	75	100	
Displacement	[in³]	[3.35]	[4.59]	[6.10]	
Flow at rated speed	l/min	215	236	300	
(theoretical)	[US gal/min]	[57]	[62]	[79]	
Torque at maximum	N•m/bar	0.88	1.19	1.59	
displacement (theoretical)	[lbf•in/1000 psi]	[530]	[730]	[970]	
Mass moment of inertia of	kg•m²	0.0060	0.0100	0.0171	
rotating components	[slug•ft²]	[0.0044]	[0.0074]	[0.0126]	
Weight (with control opt. MA)	kg [lb]	40 [88]	49 [108]	68 [150]	
Mounting (per SAE J744)		Flange SAE C			
Rotation		Right hand or Left hand rotation			
Main ports: 4-bolts split-flange	mm	25.4	25.4	25.4	
(per ISO 6162)	[in]	[1.0]	[1.0]	[1.0]	
Main port configuration	Main port configuration		Twin Ports		
Case drain ports	UNF thread (in.)	1.0625-12	1.0625–12	1.0625–12	
(SAE O-ring boss)					
Other ports		SAE O-ring boss			
Input Shafts		Splined, 21 teeth	Splined, 23 teeth	Splined, 23 teeth	

Operating Parameters

Parameters	Unit	055	075	100	
Input speed					
Minimum		400	400	400	
Rated	min ⁻¹ (rpm)	3900	3150	3000	
Maximum		4250	3350	3200	
System pressure					
Continuous			400 [5800]		
Maximum	bar [psi] 420 [6090]				
Minimum low loop pressure		10 [650]			
Suction port pressure (charge pump inlet)					
Minimum	bar (abs)		0.7 [9]		
Minimum(cold start)	[in. Hg vac.]	0.2 [24]			
Case pressure	Case pressure				
Continuous	har [nci]		3.0 [44]		
Maximum(cold start)	bar [psi]		5.0 [73]		



Technical Specifications

Fluid Specifications

Viscosity mm²/sec (cSt) [SUS]				
Minimum	7 [49]			
Recommended range	12-80 [70-370]			
Maximum	1600 [7500]			
Temperature range °C [°F]				
Minimum	-40 [-40]			
Rated	104 [220]			
Maximum intermittent	115 [240]			
Filtration				
Cleanliness	22/18/13 or higher standard ISO 4406			
Efficiency (suction line filtration)	β_{35-45} =75 (β_{10} >2)			



Operating Parameters

Input Speed

Minimum speed is the lowest input speed recommended during engine idle condition. Operating below minimum speed limits the pump's ability to maintain adequate flow for lubrication and power transmission.

Rated speed is the highest input speed recommended at full power condition. Operating at or below this speed should yield satisfactory product life.

System Pressure

System pressure is the differential pressure between high pressure system ports. It is the dominant operating variable affecting hydraulic unit life. High system pressure, which results from high load, reduces expected life. Hydraulic unit life depends on the speed and normal operating, or weighted average, pressure that can only be determined from a duty cycle analysis.

Application pressure is the high pressure relief or pressure limiter setting normally defined within the order code of the pump. This is the applied system pressure at which the drive-line generates the maximum calculated pull or torque in the application.

Maximum working pressure is the highest recommended Application pressure. Maximum working pressure is not intended to be a continuous pressure. Propel systems with application pressures at, or below, this pressure should yield satisfactory unit life given proper component sizing.

All pressure limits are differential pressures referenced to low loop (charge) pressure. Subtract low loop pressure from gauge readings to compute the differential.

Case Pressure

Under normal operating conditions, the rated case pressure must not be exceeded 3 bar (44 psi). During cold start case pressure must be kept below maximum intermittent case pressure 5 bar (73 psi). Size drain plumbing accordingly.

Caution

Possible component damage or leakage

Operation with case pressure in excess of stated limits may damage seals, gaskets, and/or housings, causing external leakage. Performance may also be affected since charge and system pressure are additive to case pressure.

Fluid Selection

Ratings and performance data are based on operating with hydraulic fluids containing oxidation, rust and foam inhibitors. These fluids must possess good thermal and hydrolytic stability to prevent wear, erosion, and corrosion of motor components. Never mix hydraulic fluids of different types.

Fire resistant fluids are also suitable at modified operating conditions. Please see *Hydraulic Fluids and Lubricants Technical Information*, **520L0465**, for more information.

The following hydraulic fluids are suitable:

- Hydraulic Oil DIN 51 524-2 HLP
- Hydraulic Oil DIN 51 524-3 HVLP
- SAE J183 API CD, CE and CF



Operating Parameters

Temperature and Viscosity

The high temperature limits apply at the hottest point in the transmission, which is normally the motor case drain. The system should generally be run at or below the quoted **rated temperature**. The **maximum intermittent temperature** is based on material properties and should never be exceeded.

Cold oil will generally not affect the durability of the transmission components, but it may affect the ability of oil to flow and transmit power; therefore temperatures should remain 16 °C [30 °F] above the pour point of the hydraulic fluid. The **minimum temperature** relates to the physical properties of component materials. Size heat exchangers to keep the fluid within these limits. Danfoss recommends testing to verify that these temperature limits are not exceeded.

For maximum efficiency and bearing life, ensure the fluid viscosity remains in the **recommended range**. The **minimum viscosity** should be encountered only during brief occasions of maximum ambient temperature and severe duty cycle operation. The **maximum viscosity** should be encountered only at cold start.

Filtration System

To prevent premature wear, ensure only clean fluid enters the hydrostatic transmission circuit. A filter capable of controlling the fluid cleanliness to ISO 4406 class 22/18/13 (SAE J1165) or better, under normal operating conditions, is recommended. These cleanliness levels can not be applied for hydraulic fluid residing in the component housing/case or any other cavity after transport.

The selection of a filter depends on a number of factors including the contaminant ingression rate, the generation of contaminants in the system, the required fluid cleanliness, and the desired maintenance interval. Filters are selected to meet the above requirements using rating parameters of efficiency and capacity.

Filter efficiency can be measured with a Beta ratio¹ (β X). For simple suction-filtered closed circuit transmissions and open circuit transmissions with return line filtration, a filter with a β -ratio within the range of $\beta_{35-45} = 75$ ($\beta_{10} \ge 2$) or better has been found to be satisfactory.

Reservoir

The hydrostatic system reservoir should accommodate maximum volume changes during all system operating modes and promote de-aeration of the fluid as it passes through the tank. A suggested minimum total reservoir volume is 5/8 of the maximum charge pump flow per minute with a minimum fluid volume equal to ½ of the maximum charge pump flow per minute. This allows 30 seconds fluid dwell for removing entrained air at the maximum return flow. This is usually adequate to allow for a closed reservoir (no breather) in most applications.

Locate the reservoir outlet (charge pump inlet) above the bottom of the reservoir to take advantage of gravity separation and prevent large foreign particles from entering the charge inlet line. Position the reservoir inlet (fluid return) to discharge below the normal fluid level, toward the interior of the tank. A baffle (or baffles) will further promote de-aeration and reduce surging of the fluid.

Case Drain

A case drain line must be connected to one of the case outlets (L1 or L2) to return internal leakage to the system reservoir.



System Design Parameters

Shaft Loads

The table below indicates the bearing life in B₁₀ hours. These data are based on the condition where the pump is operated with system pressure at 240bar[3500 psi], input speed at 1800RPM, with max. displacement and no external thrust/radial shaft loads. Nearly equal amounts of foward vs. reverse swashplate operation is experienced. The charge pump is of standard displacement and is a standard charge pressure pump.

T90 piston pumps are designed with bearings that can accept some external radial and thrust loads. The external shaft radial load limits are a function of the load position and orientation, and operating conditions of the motor.

The maximum allowable radial load (Re) is based on the maximum external moment (Me) and the distance (L) from the mounting flange to the load. It may be determined using the following table and formula.

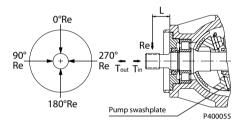
Formula:

Re = Me / L

Shaft life

	Parameter	Bearing life – B ₁₀ hours
	55	22 090
	75	22 970
ĺ	100	22 670

Radial/thrust load position



All external shaft loads affect bearing life. In applications where external shaft loads cannot be avoided, minimize the impact by positioning the load at 90° or 270° as shown in the figure.

Contact your Danfoss representative for an evaluation of unit bearing life, if

- you have continuously applied external loads exceeding 25 % of the maximum allowable radial load (Re)
- or the pump swashplate is positioned on one side of center all or most of the time.
- bearing life B₁₀ is critical.

Use of tapered output shafts or clamp-type couplings is recommended where radial shaft loads are present.

Allowable external shaft load

Parameters	055	075	100
External moment (Me)	101	118	126
N•m [lbf•in]	[893]	[1043]	[1115]
Maximum shaft thrust in (T _{in})	3340	4300	5160
N [lbf]	[750]	[996]	[1160]
Maximum shaft thrust in (T _{out})	910	930	1000
N [lbf]	[204]	[209]	[224]



System Design Parameters

Shaft Availability and Torque Ratings

Shaft availability and torque ratings

Shaft description	055	075	100
21 teeth 16/32 pitch spline	1130 [10 000]	_	_
23 teeth	[10 000]	1580	1580
16/32 pitch spline	_	[14 000]	[14 000]

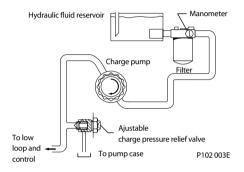
Filtration

Suction filtration

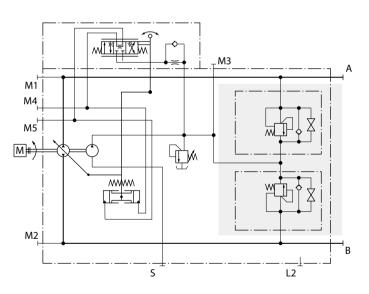
The suction filtration is placed in the circuit between the reservoir and inlet to the charge pump, as shown below.

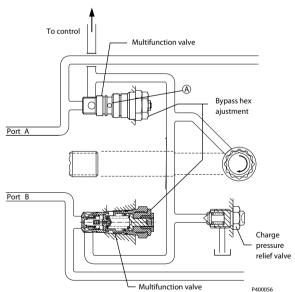
Filter with block alarm is recommended

Suction filtration



Multi-Function Valves







Features

Charge Pump

Charge flow is required on all Series T90 pumps applied in closed circuit installations. The charge pump provides flow to make up internal leakage, maintain a positive pressure in the main circuit, provide flow for cooling and filtration, replace any leakage losses from external valving or auxiliary systems, and to provide flow and pressure for the control system.

Many factors influence the charge flow requirements and the resulting charge pump size selection. These factors include system pressure, pump speed, pump swashplate angle, type of fluid, temperature, size of heat exchanger, length and size of hydraulic lines, control response characteristics, auxiliary flow requirements, hydrostatic motor type, etc. When initially sizing and selecting hydrostatic units for an application, it is frequently not possible to have all the information necessary to accurately evaluate all aspects of charge pump size selection.

Recommend charge pump sizes and speed limits

Charge pump size cm³ [in³]	Rated speed min ⁻¹ (rpm)
20 [1.20]	3600

Manual Displacement Control (MDC)

Operation

The manual displacement control converts a mechanical input signal to a hydraulic signal that tilts the cradle swashplate through an angular rotation varying the pump's displacement from full displacement in one direction to full displacement in the opposite direction.

The manual displacement control has a mechanical feedback mechanism which moves a servo valve in the proper relationship to the input signal and the angular position of the swashplate. The control is designed so that the angular rotation of the swashplate is proportional to the mechanical input signal. The control is designed with an internal override mechanism which allows the mechanical input to be moved at a faster rate than the movement of the swashplate without damage to the control.

Features and benefits of the manual displacement control:

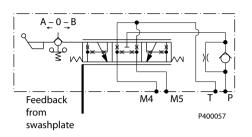
- Precision parts provide repeatable, accurate displacement settings with a given input signal.
- The manual displacement control is a high gain control: With only small movement of the control handle (input signal), the servo valve moves to full open position porting maximum flow to the servo cylinder. This is a high response system with low input force.
- The integral override mechanism allows rapid changes in input signal without damaging the control mechanism.
- The double-acting servo piston is coupled to a spring centering mechanism. The servo control valve is spring centered such that with no input signal the servo valve is open centered and thus no fluid is ported to the servo cylinder.
- Benefits:
 - Pump returns to neutral after prime mover shuts down.
 - Pump returns to neutral if external control linkage fails at the control handle or if there is a loss of charge pressure.



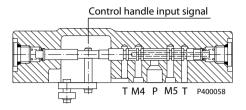
Features

Manual Displacement Control (MDC)

Manual displacement control schematic



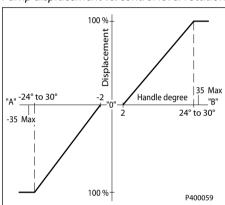
Cross-section



External control handle requirements

- Torque required to move handle to maximum displacement is 0.68 to 0.9 N·m [6 to 8 lbf·in].
- Torque required to hold handle at given displacement is
 0.34 to 0.57 N·m [3 to 5 lbf·in].
- Torque required to overcome the override mechanism is 1.1 to 2.3 N·m [10 to 20 lbf·in] with the maximum torque required for full forward to full reverse movement.
- Maximum allowable input torque is 17 N•m [150 lbf•in].

Pump displacement vs. control lever rotation



Control lever rotation range

a	0.5° - 4.5°
b	24° - 30°

Volumetric efficiencies of the system will have impacts on the start- and end inputcommands.

Pump output flow direction and control lever rotation

Input shaft rotation	CW		cc	:W
Handle rotation	A CCW B CW		A CCW	B CW
Port A flow (M1)	Out	In	In	Out
Port B flow (M2)	ln	Out	Out	In
Servo cylinder	M5	M4	M5	M4

Refer to Installation drawings for handle connection requirements



Control

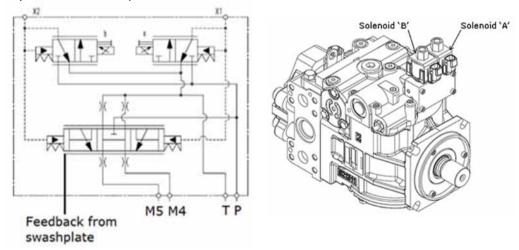
High Current Electric Displacement Control (HCEDC) Option PH and PJ

Operation

The HCEDC uses two solenoid operated, proportional-pressure reducing valves to control the pilot pressure to a 4-way servo valve, which ports hydraulic pressure to either side of a double acting servo piston. The servo piston tilts the cradle swashplate, thus varying the pump's displacement from full displacement in one direction to full displacement in the opposite direction. Each solenoid valve acts independently for forward or reverse operation; therefore, the electronic controller must be able to accommodate two independent pilot valve signal outputs.

The control has a mechanical feedback mechanism which moves the servo valve in relation to the input signal and the angular position of the swashplate. The electrical displacement control is designed so the angular rotation of the swashplate (pump displacement) is proportional to the electrical input signal. Swashplate position changes due to load variation are sensed by feedback linkage system connected to the swashplate and control valve. This will activate the valve and supply pressure to the servo piston, maintaining the swashplate in its commanded position. The solenoids are equipped with manual override capability thereby allowing the pump to be commanded to maximum angle in either direction. This is done by depressing the plunger on the top of the solenoid. Manual operation of the control override is intended for system troubleshooting only.

High current electric displacement control schematic



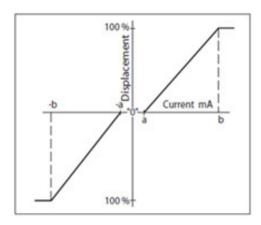
Electric Characteristics

	(CCW) AS SEEN FROM SHAFT		(CW) AS SEEN	FROM SHAFT	
Active Solenoid	Α	В	Α	В	
Pressurized port	X1	X2	X1	X2	
System port A flow	ln	Out (M1)	Out (M1)	ln	
System port B flow	Out (M2)	In	In	Out (M2)	
Servo port active	2 (M5)	1 (M4)	2 (M5)	1 (M4)	
Options	Р	PH		ני	
Starting current "a"	350mA		"a" 350mA 178mA		mA
Maximum current "b"	850mA		440	mA	



Control

High Current Electric Displacement Control (HCEDC) Option PH and PJ -Continued



	PH	PJ	
Maximum current	850mA	440mA	
PWM frequency	100 - 200 Hz		
Coil resistance @ 20 ℃	9.0 Ω 35.6 Ω		

The Option PJ coils have an IP 69 K environmental protection rating. The coils include a uni-directional, polarity diode which protects downstream electronic components from power surges originating from the coil. Therefore, care must be taken to not reverse the "+" and "-" terminals. Failure to do so will damage the diode and render the coil unusable. The coils have a "1" and "2" molded in the connector for proper identification of the poles.

POSITIVE 1 © Pin identifier Pin iden

The Option PH (12V) and Option PJ (24V) controls can be distinguished by the color of the shroud. The 24V Option PJ has a yellow shroud while the 12V Option PH has a blue shroud.



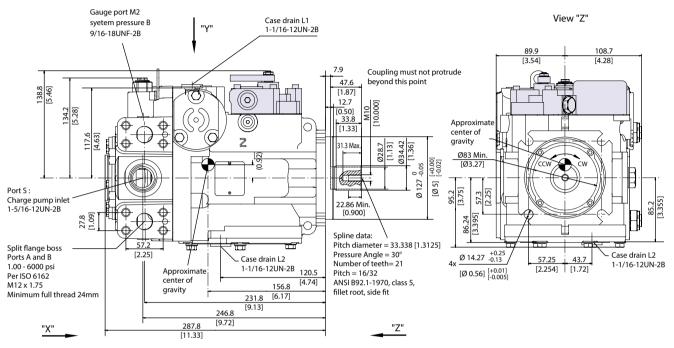


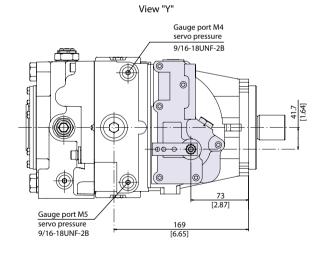
Installation Drawings

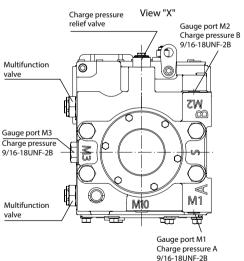
mm [in]

Size 055

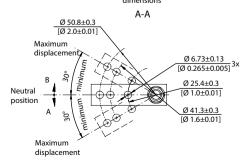
Manual Displacement Control(MDC), endcap twin ports







Manual displacement control handle dimensions



P400060

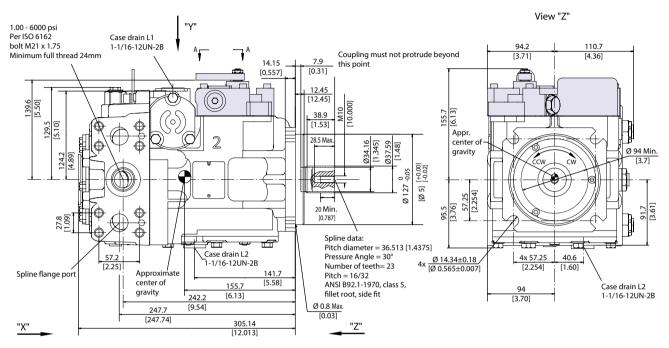


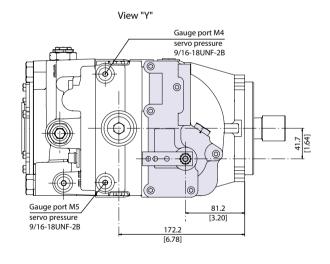
Installation Drawings

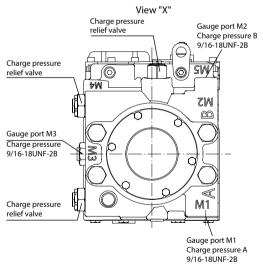
mm [in]

Size 075

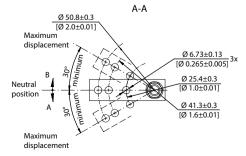
Manual Displacement Control(MDC), endcap twin ports







Manual displacement control handle dimensions



P400061

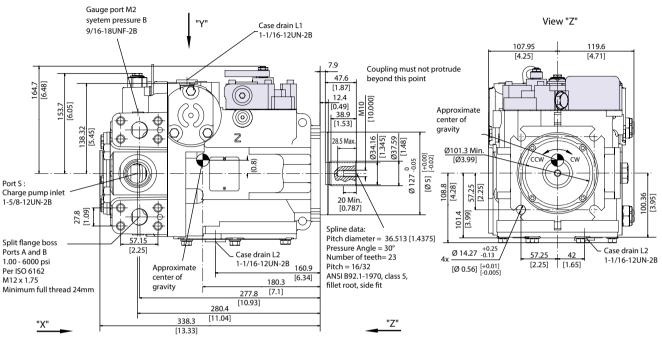


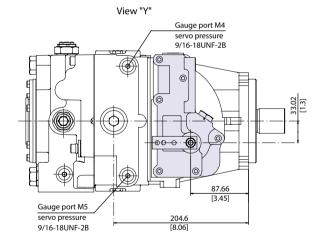
Installation Drawings

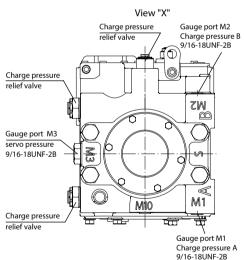
mm [in]

Size 100

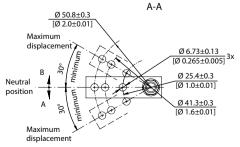
Manual displacement control (MDC) endcap twin ports







Manual displacement control handle dimensions



P400062



General Description

Series 90 Family of Motors

Series 90 motors also use the parallel axial piston/slipper design in conjunction with a fixed or tiltable swashplate. They can intake/discharge fluid through either port; they are bidirectional. They also include an optional loop flushing feature that provides additional cooling and cleaning of fluid in the working loop.

Features

Parameter	055 MF 075 MF		100 MF	
Types of mounting (SAE flange	SAE C SAE C		SAE C	
size per SAE J744)				
Port connections	Twin, axial	Twin, axial	Twin	
Output shaft options	Spline, tapered, straight	Spline, tapered, straight	Spline, tapered, straight	
Loop flushing	•	•	•	
Speed sensor	0	0	0	

Standard

Specifications

Parameter	055 MF	075 MF	100 MF Fixed	
Swashplate	Fixed	Fixed		
Max. displacement cm³/rev [in³/rev]	55 [3.35]	75 [4.57]	100 [6.10]	
Maximum corner power kW [hp]	164 [220]	176[236]	224 [300]	
Theoretical torque N•m/bar [lbf•in/1000 psi]	0.88 [530]	1.19 [730]	1.59 [970]	
Weight kg [lb]	22 [49]	26 [57]	34 [74]	
Mass moment of inertia kg·m² [slug·ft²]	0.0060 [0.0044]	0.0100 [0.0074]	0.0171 [0.0126]	

Operating Parameters

Parameter	Unit	055 MF	075 MF	100 MF
Speed limits				
Continuous (max. disp.)	min ⁻¹ (rpm)	3900	3150	3000
Maximum (max. disp.)		4250	3350	3200
System pressure				
Continuous	la a u far att	400 [5800]		
Maximum	bar [psi]	420 [6090]		
Flow ratings				
Rated	l/min [US gal/min]	215 [57]	226 [62]	200 [70]
(max. disp., rated speed)		215 [57]	236 [62]	300 [79]
Maximum		224 [62]	251 [66]	220 [05]
(max. disp., max. speed)		234 [62]	251 [66]	320 [85]
Case pressure				
Continuous	hau (nai)	3.0 [44]		
Maximum (cold start)	bar [psi]	5.0 [73]		

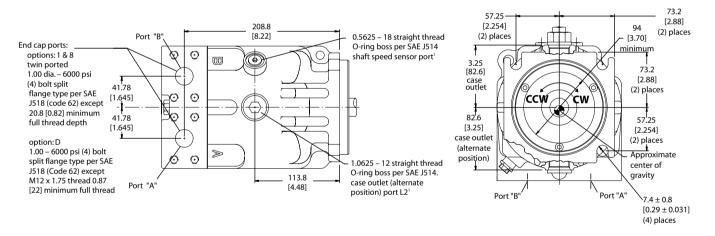
O Optional

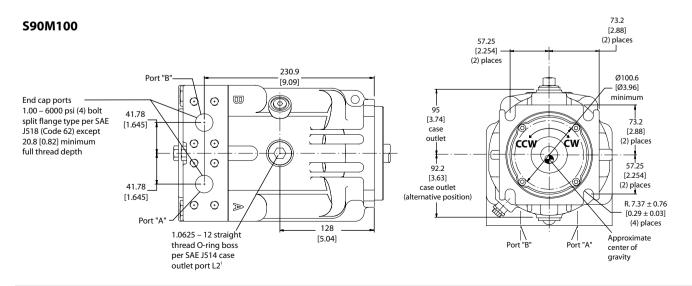


Installation Drawings

mm [in] S90M55 57.25 [2.254] 73.2 [2.88] (2) places (2) places 189.5 84.8 [3.34] [7.46] minimum 0 0 \Box Port "B" 732 73.9 [2.91] [2.88] 41.78 case outlet (2) places [1.645] 0 41.78 0 \odot 57.25 [2.254] [1.645] 76.2 [3.00] case outlet (2) places End cap ports alternate position 1.00 in dia. 6000 psi 1 ≫ (4) bolt split flange type per SAE J518 0 R. 7.4 ± 0.8 [0.29 ± 0.03] 0.875 - 14 straight thread O-ring boss (code 62) except per SAE J514 case (4) places 20.8 [0.82] minimum 103.6 outlet port L21 Port "A" [4.08] full thread depth Port "A" Approximate Port "B" center of gravity

S90M75









Products we offer:

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- Closed Circuit Axial Piston Pumps and Motors
- Displays
- Electrohydraulic Power Steering
- Electrohydraulics
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